

HYDROSTATIC STEERING UNITS

GENERAL APPLICATION AND SPECIFICATION INFORMATION

APPLICATION

(SIZING AND STEERING SYSTEM DESIGN PROCESS)

STEP ONE:

Calculate approximate kingpin torque (M_L).

$$M_L = G \cdot \mu \sqrt{\frac{B^2}{8} + \ell^2}$$

Note: Double M_L if steered wheels are powered.

M_L = Kingpin torque in *in-lb* [daNm].

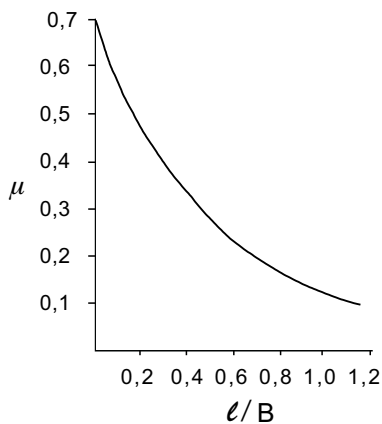
G = Vehicle weight on steered axle *lbs* [daN] (use maximum estimated overload weight).

μ = Coefficient of friction (use Chart 1, dimensionless) determined by ℓ/B (see Diagram 1).

B = Nominal width of tyre print *in.* [m] (see Diagram 1).

ℓ = Kingpin offset. The distance between tyre centerline intersection at ground and kingpins centerline intersection at ground in *in.* [m] (see Diagram 1).

Chart 1 1



Rubber tyres on dry concrete.

Diagram 1 1

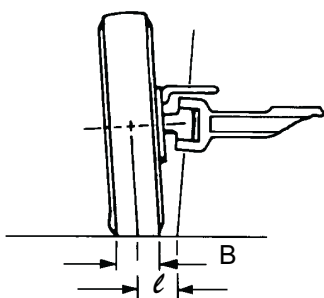
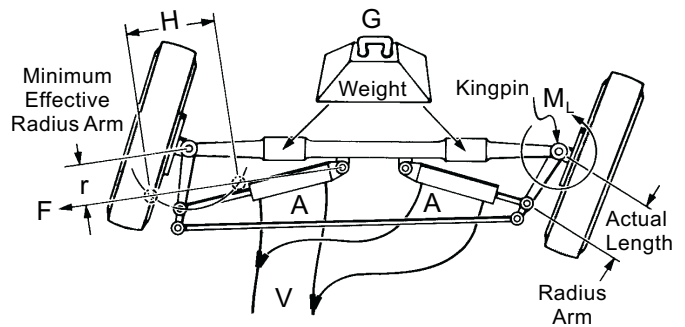


Diagram 1 2



STEP TWO:

Calculate approximate cylinder; force-area-stroke-volume.

FORCE

$$F = \frac{M_L}{r}$$

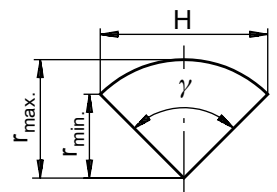
F = Force required *lbs* [daN] to steer axle.

M_L = Kingpin torque in *in-lb* [daNm] from step one. Double M_L if steered wheels are powered.

r = Effective radius Arm *in.* [mm] is the minimum distance from the centerline of the cylinders minimum and maximum stroke points parallel to the kingpin center pivot. This is not the physical length of the radius Arm (see Diagram 1 2 and Chart 1 2).

Chart 1 2

$$r_{\min.} = r_{\max.} \cdot \cos \frac{\gamma}{2}$$



STROKE

H = Stroke *in.* [cm].

Calculate stroke of cylinder using Diagram 1 2 and Chart 1 2 as shavt.

$$H = 2 r_{\max.} \cdot \sin \frac{\gamma}{2}$$

AREA

$$A = \frac{F}{\Delta P}$$

A = Cylinder area for axle cylinder set *in.*² [cm²].

F = Force required from step two force formula *lbs* [daN].

ΔP = Hydraulic pressure *PSI* [bar] use following percentage of relief valve setting by amount of load on steered axle. Severe load 25% - medium load 55% - no load 75%.

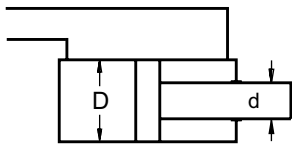
DIAMETER

After the cylinder set area is determined, the cylinder diameter can be calculated.

D = Inside diameter of cylinder in. [cm].
d = Road diameter of cylinder in. [cm].

Choose type of cylinder arrangement and formula shown for that type.

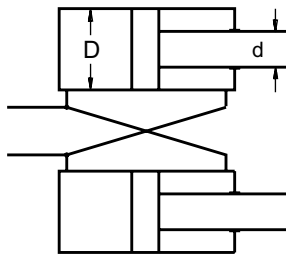
Differential Cylinder



$$D = \sqrt{\frac{4A}{\pi} + d^2}$$

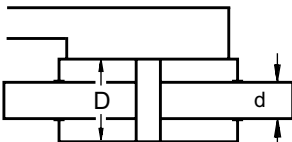
Note: $\left(\frac{d}{D}\right)^2 \leq 0,15$

Cross Connected Cylinders



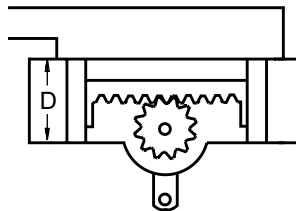
$$D = \sqrt{\frac{2A}{\pi} + \frac{d^2}{2}}$$

Balanced Cylinder



$$D = \sqrt{\frac{4A}{\pi} + d^2}$$

Opposed Cylinder



$$D = \sqrt{\frac{4A}{\pi}}$$

VOLUME $V = H \cdot A$

V = Volume. The total amount of oil required to move the cylinder rod(s) through the entire stroke in.³ [cm³].

H = Stroke in. [cm].

A = Area in.² [cm²].

Note: For differential cylinders it is important to calculate average cylinder volume for step three using below formula.

$$V_{avg.} = H \cdot \frac{\pi}{4} (2 \cdot D^2 - d^2)$$

STEP THREE:

Selecting displacement of hydrostatic steering unit.

At this point determine number of steering wheel revolutions desired for your application to steer the wheels from one side to the other (lock to lock). Depending on the type of vehicle and its use, this will vary from 3 to 5 turns.

DISPLACEMENT $V_D = \frac{V}{n}$

V_D = Displacement in.³/rev. [cm.³/rev.]

V = Volume of oil in.³ [cm³].

n = Steering wheel turns lock to lock.

After completing the above displacement calculation, choose the closest standard hydrostatic steering unit in displacement size that incorporates circuitry you require. Recalculate the number of steering wheel turns using the displacement of selected standard hydrostatic steering unit outlined above. Use the formula shown below.

$$n = \frac{V}{V_D}$$

V = Volume of oil in.³ [cm³].

n = Steering wheel turns lock to lock.

Note: For differential cylinders applications the cylinder volume will be different for left and right turns - this means the value n (steering wheel turns lock to lock) will vary when turning to the left or right.

STEP FOUR:

Calculate approximate minimum and maximum steering circuit flow requirements.

$$Q = \frac{V_D \cdot N}{\text{Unit Conversion for Imperial or [1000] Metric}}$$

Q = Steering circuit flow GPM [lpm].

V_D = Unit displacement in.³/rev. [cm.³/rev.]

N = Steering wheel input speed RPM.

Recommended steering speed is 50 to 100 RPM.

Many variables are involved in sizing the pump. We suggest that the manufacturer test and evaluate for desired performance.

GENERAL INFORMATION

FLUID DATA:

To insure maximum performance and life of the Hydrostatic steering units, use premium quality hydraulic oils. Fluids with effective quantities of anti-wear agents or additives are highly recommended. If using synthetic fluids consult the factory for alternative seal materials.

Viscosity

Viscosity at normal operating temperature should be approx. 100 SUS [20 mm²/s]. Viscosity range 60 - 1500 SUS [10 - 300 mm²/s].

Temperature

Normal operating temperature range from +85°F [+30°C] to 140°F [+60°C].

Minimum operating temperature -40°F [-40°C].

Maximum operating temperature +176°F [+80°C].

Note: Extended periods of operation at temperature of 60°C and above will greatly reduce life of oil due to oxidation and shorten life of product.

STEERING UNIT

Filtration

The maximum degree of contamination per ISO 4406 or CETOP RP is:

- 20/17 open center units
- 19/16 closed center and load sensing
- 16/12 priority valves

Return line filtration of 25 μm nominal (40 - 50 μm absolute) or finer is recommended.

In extremely dusty conditions filtration of 10 μm absolute should be used.

START UP

All air must be purged from system before operating unit. It is extremely important that any external lines or units with load sensing or priority feature be completely bled. Lines going to and from cylinders as well as lines to and from pump be purged of all air. It is recommended that a 10-15 μm filter be used between pump and steering unit before start up.

MOUNTING UNITS

All hydrostatic steering units should be installed for ease of access. It is recommended that the steering unit be located outside the vehicle cabin.

It is important that no radial axial load be applied to the hydrostatic steering unit input shaft. Any or all radial and axial loads must be absorbed by the steering column or other operating device supplied by the vehicle manufacture. Ports on the steering cylinder(s) should face upward to prevent damage.

During installation of the hydrostatic steering unit, cleanliness is of the utmost importance. Pipe plugs should be left in place during mounting and only removed when hydraulic lines are to be connected.

CONVERSIONS

to convert inches and millimeters:

- 1 in. = 25,4 mm
- 1 mm = .03973 in.

to convert gallons per minutes and liters per minutes:

- 1 GPM = 3,785 lpm
- 1 lpm = .2642 GPM

to convert pounds per square inch and bar:

- 1 PSI = 0,0689 bar
- 1 bar = 14,51 PSI

to convert pounds-inch and newton-meters:

- 1 in - lb = 0,113 Nm
- 1 Nm = 8.85 in - lb

TORQUE TIGHTENING VALUES

Fluid connections

Fluid connection	Max. tightening torque in - lb [daNm]			
	metal edge	copper washer	aluminum washer	O - ring
7/16 - 20 UNF				180[2]
9/16 - 18 UNF				440[5]
3/4 - 16 UNF				530[6]
7/8 - 14 UNF				620[7]
G 1/4	350[4]	180[2]	270[3]	
G 3/8	530[6]	180[2]	440[5]	
G 1/2	885[10]	270[3]	710[8]	
G 3/4	1400[16]	440[5]	1150[13]	
M 10 x 1	350[4]	180[2]	270[3]	
M 18 x 1,5	620[7]	180[2]	440[5]	
M 22 x 1,5	885[10]	270[3]	710[8]	

Mounting bolts

Mounting bolts	Tightening torque in - lb [daNm]
3/8 - 16 UNC	230 ÷ 310 [3,0 ± 0,5]
M 10 x 1	540 ÷ 620 [6,5 ± 0,5]
M 10	230 ÷ 310 [3,0 ± 0,5]